

Rail Fastening Systems and Buckling of Continuous Welded Rails

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Today's topics

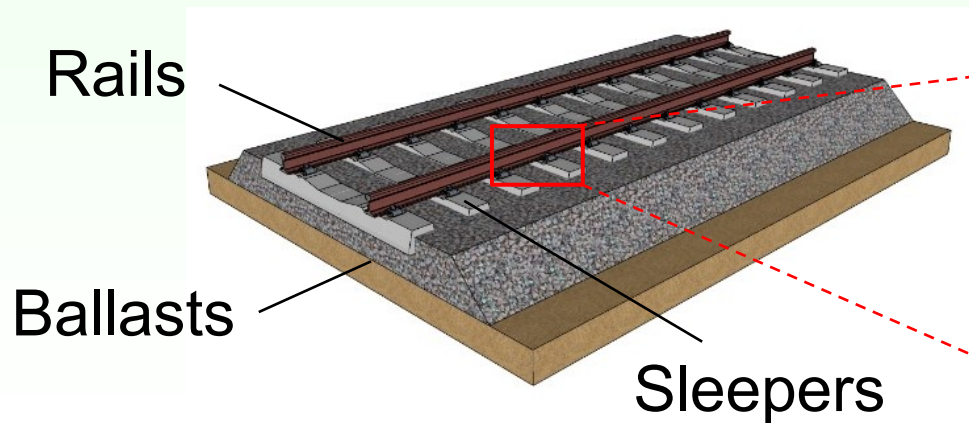
- 1. Rail fastening systems**
- 2. Buckling of continuous welded rails**

1. Rail fastening systems

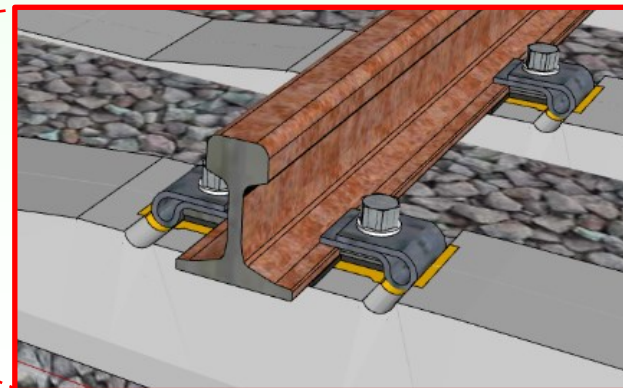
What are rail fastening systems?

Rail fastening systems are the devices

- used to **fasten two rails to sleepers**.
- that **keep the gauge** (distance between rails).
- that **resist various loads and vibrations** from the trains, and transfer them to the supporting structures.



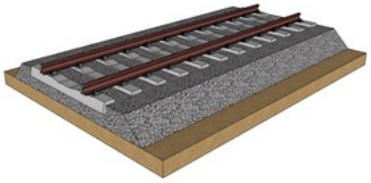
Rail fastening systems



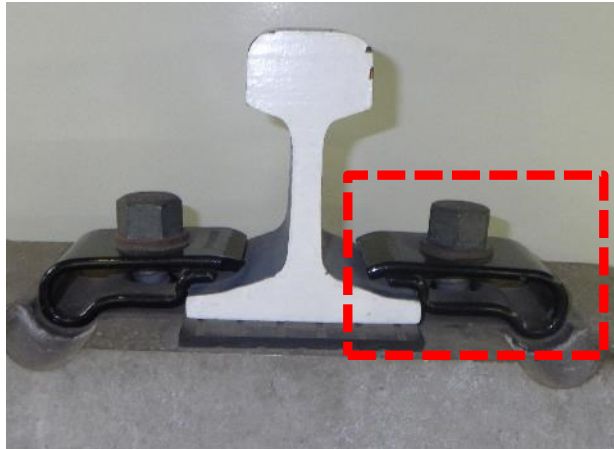
Rail clips
Rail pads
Bolts & Nuts

Major rail fastenings (Domestically developed in Japan)

Ballasted tracks

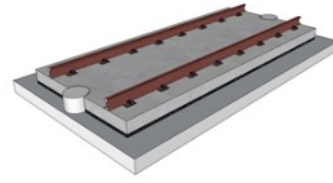


Wooden sleepers
(Type F)



PC sleepers
(Type 5N)

Slab tracks



Slabs
(Type 8 for slab)

Tracks on Steel bridges

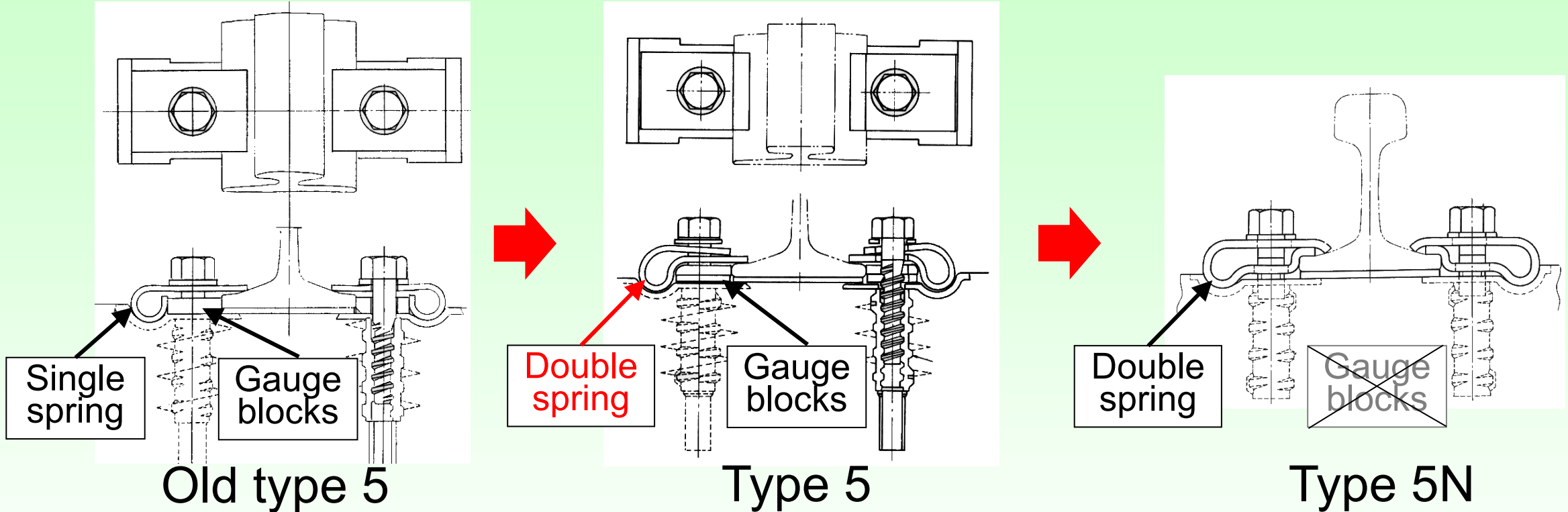


Steel bridges
(Type 5 for bridge)

In domestic products, it is common to use **plate spring-type rail clips** for rail fastenings.

Transition of plate spring-type rail fastenings

※Using the type 5 as an example

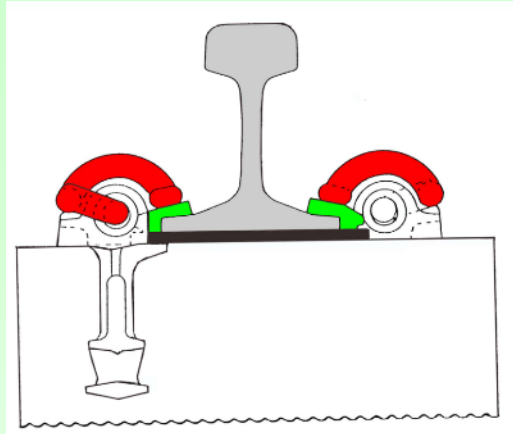


Strengthen plate spring
(Double springing)

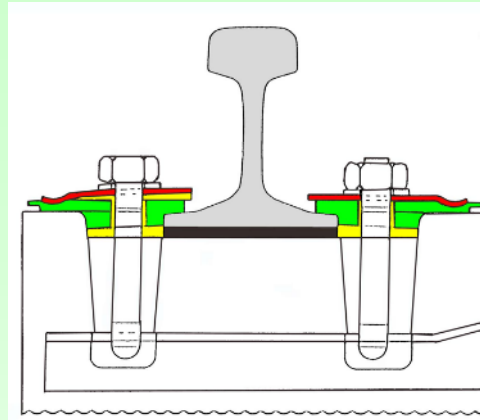
Simplification of components

After **multiple improvements**, it has evolved into its current form.

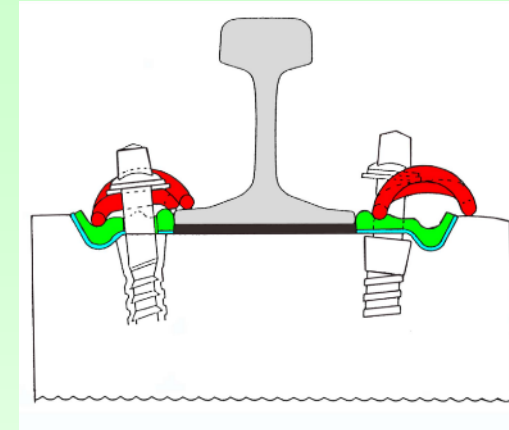
Major rail fastenings (Developed outside of Japan)



Pandrol



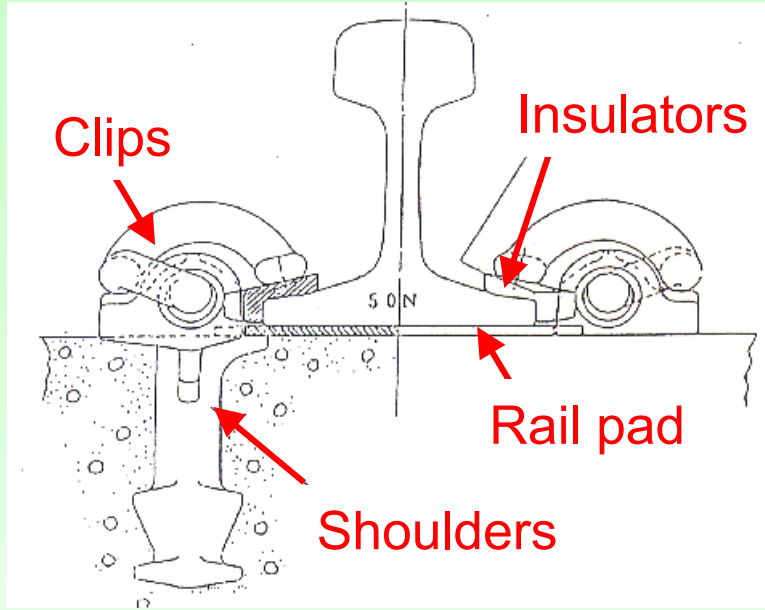
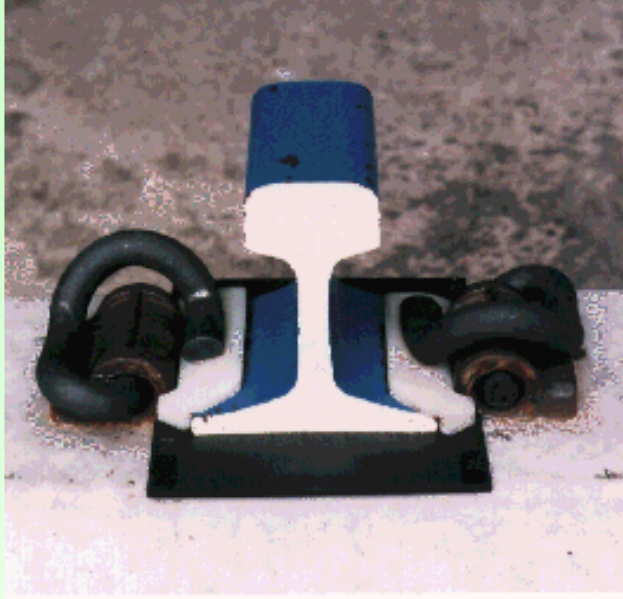
Nabla



Vossloh

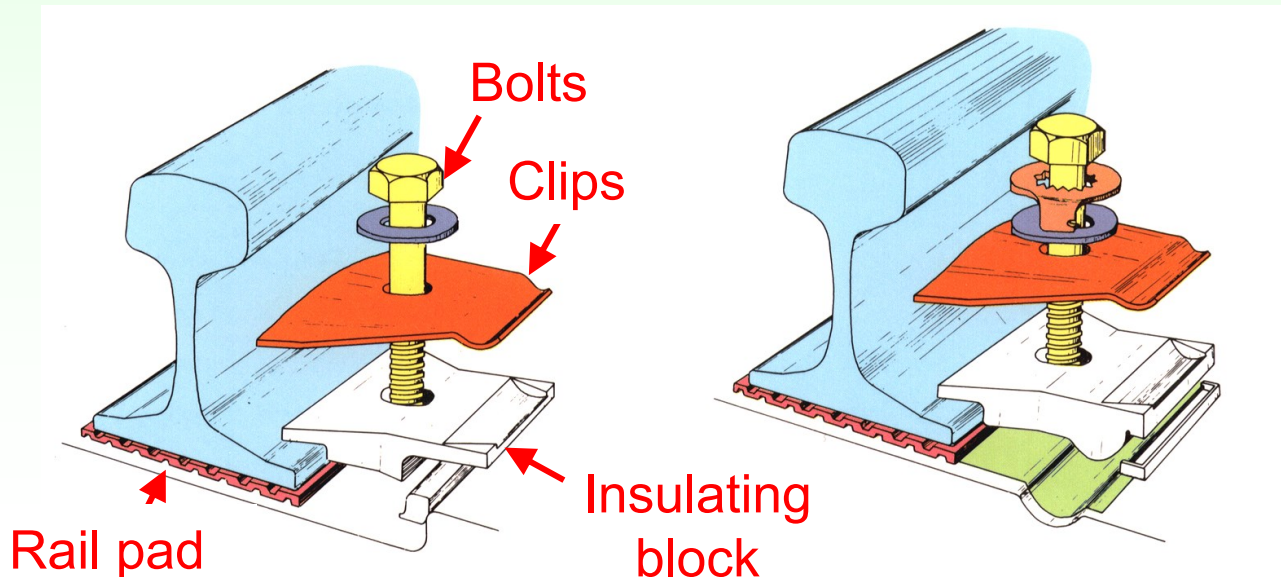
For decades, rail fastenings **developed outside of Japan** have been widely used in Japan.

Pandrol




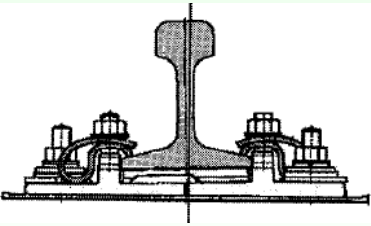










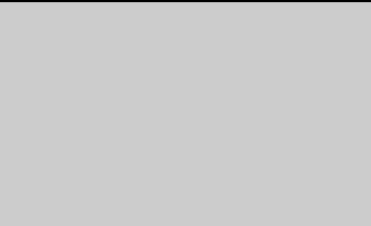

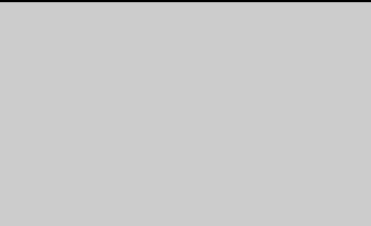

- ◆ Developed by Pandrol, UK
- ◆ Introduced to Japan in the 1990s
- ◆ Using **bar-type rail clip**
- ◆ Clip stands out for not loosening

Nabla



- ◆ Developed in France for TGV use
- ◆ Using **plate-type rail clips** and **insulating block**
- ◆ High tightening torque and less bolt loosening
- ◆ High electrical insulation

An example of classification for rail fastenings

Wooden sleeper

PC sleeper

Slab

Steel bridge

Rail supports

Base plate
Double spring
Resin
Others

Lateral loads supports

There are more than **100 types** in use within Japan !

Why are there so many types of rail fastenings ?

This is because

- they are developed in **response to the transition of track structures.**

Combinations

- **Rails:** 30kg, 37kg, 40kgN, 50kgN, 60kg
- **Supporters:** Wooden or PC sleepers, Slabs, Steel bridges
- **Line:** High-speed line(Shinkansen), Conventional line
- **Alignment:** Sharp curve, Gentle curve, Straight
- **Others:** Rail joints (EJ, IJ), Level crossings, etc.

- different functions are required for them depending on track structures.

Required functions of rail fastenings

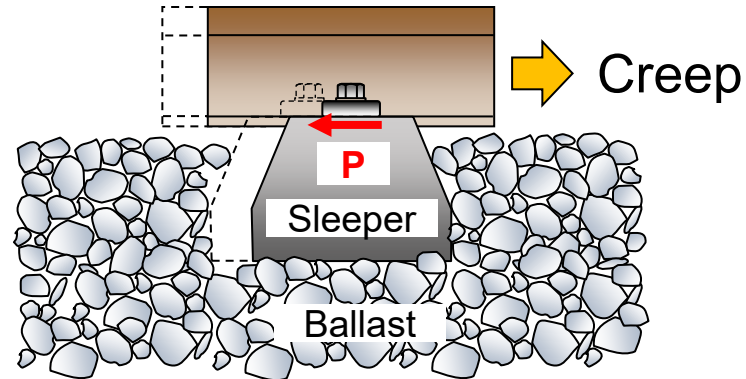
- ① Fixing the rails to the support structures
- ② Buffering impact forces transmitted from the rails
- ③ Dispersing loads transmitted from the vehicle to the rails
- ④ Resisting horizontal forces transmitted from the rails
- ⑤ Resisting rail inclination
- ⑥ Resisting horizontal plate rotation of the rails
- ⑦ **Resisting rail creep**
 - ➔ This function is completely different between ballasted track and slab track.
- ⑧ Allowing vertical and lateral adjustment of the rails
- ⑨ Electrically insulating between the rail and the support structure
- ⑩ Capable of mass production and cost-effective
- ⑪ Reducing vibrations transmitted from the rail to the support structure

Differences in creep resistances of rail fastenings between ballasted tracks and slab tracks

Ballasted tracks



Rail creeps with sleepers

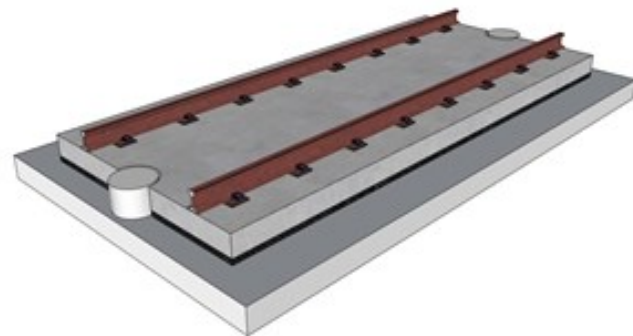


【Creep resistance P 】

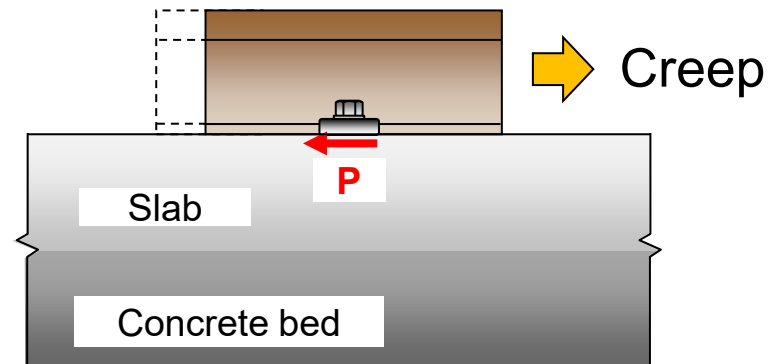
- Greater than longitudinal ballast resistance

P is about 15 kN/m

Slab tracks



Rail creeps on slabs

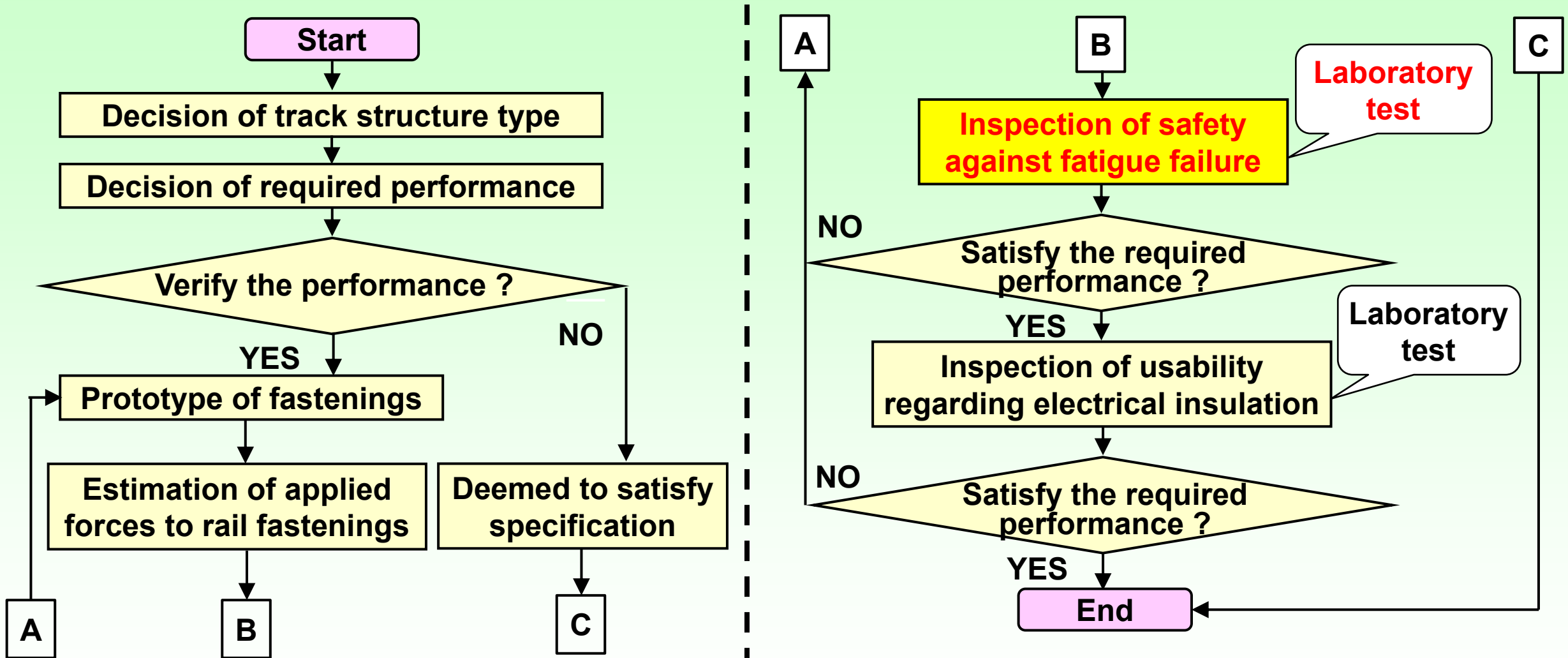


【Creep resistance P 】

- Reducing the forces transmitted to the support structures

P is about 5 kN/m

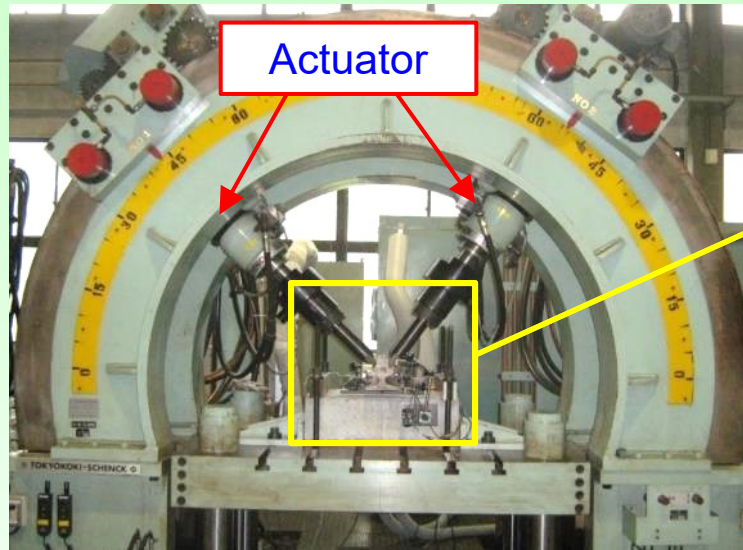
Basic flow of design for rail fastening systems



The Japanese design standard for railway structures requires the **safety to be verified** by the **laboratory test**.

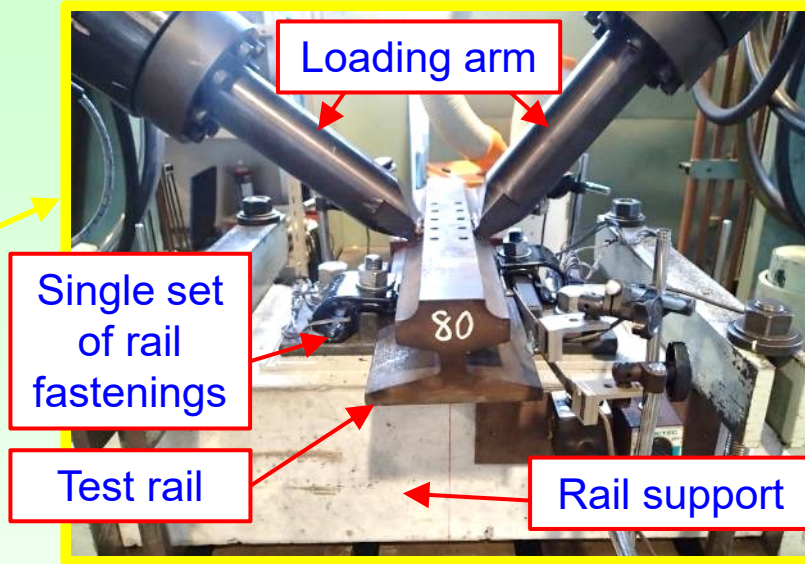
Laboratory test to inspect the safety against fatigue failure

Two directional load test



Inside of gauge

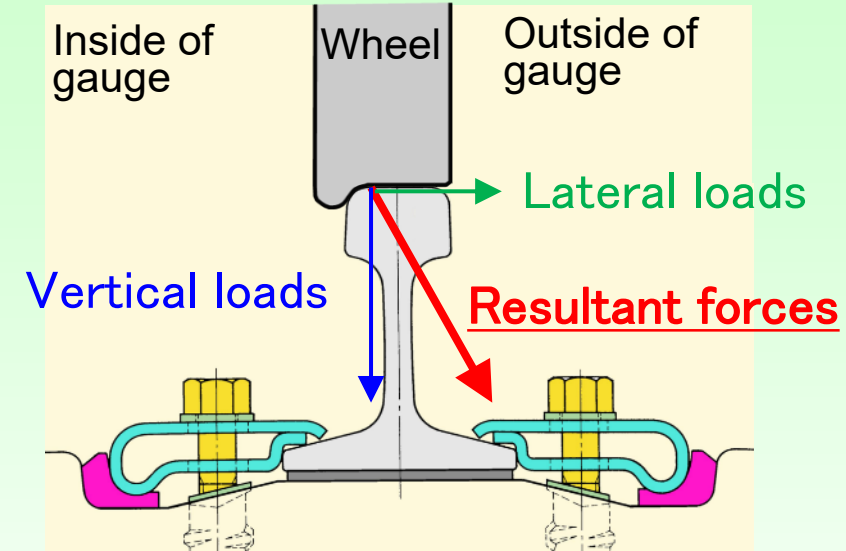
Outside of gauge



Single set of rail fastenings

Test rail

Rail support

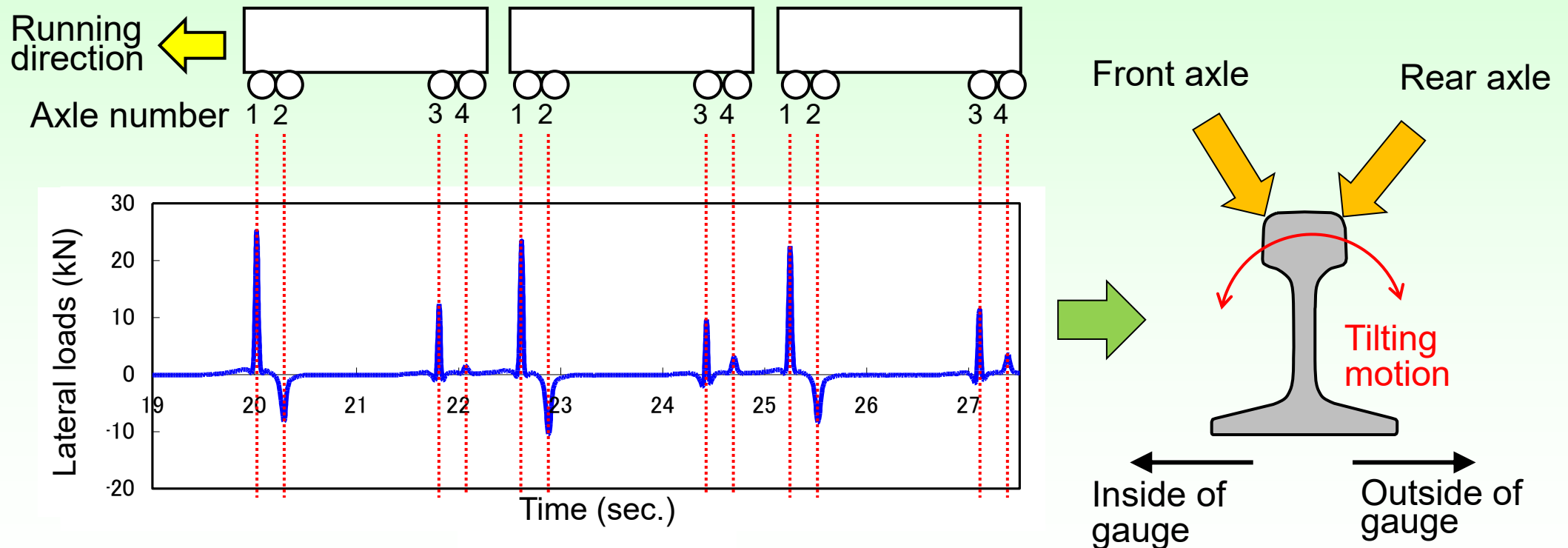


Resultant forces of vertical and lateral loads are **alternately applied to the test rail from inside and outside directions** of the gauge.

✂ **Two directional loading** is a unique approach in Japan.

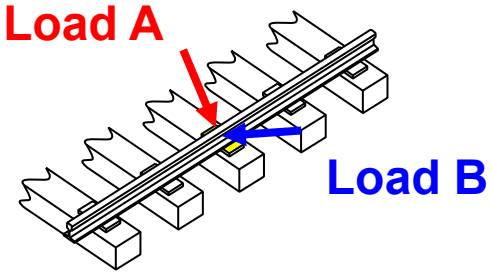
Why loading from two directions?

Past on-site measurements in curved sections show that **lateral loads occur that tilt the rail alternately to the outside and inside of the gauge**, when the front and rear axles of the train's bogie pass.



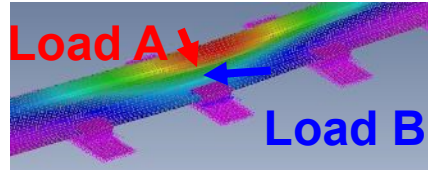
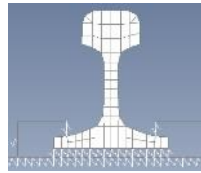
Measured lateral force at sharp curve (R=160m)

Estimation of applied loads used in the test

Design loads	Load type	Alignment	Values	
			Load A (Max, rarely)	Load B (Avg, often)
	Vertical loads	Tangent, Curved	98 kN	86 kN
	Lateral loads	Tangent, $R \geq 800\text{m}$	30 kN	15 kN
		$600\text{m} \leq R < 800\text{m}$	45 kN	23 kN
		$R < 600\text{m}$	60 kN	30 kN

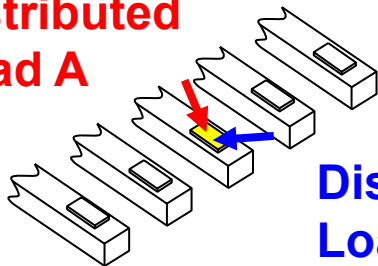
Values are for meter-gauged railway line

FEM analysis



Distributed design loads

Distributed Load A

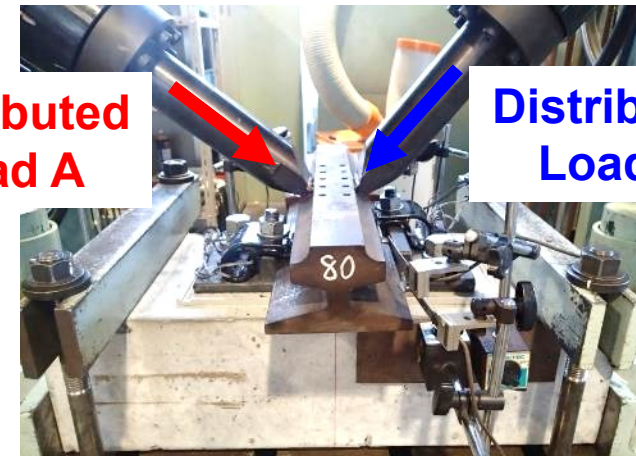


Distributed Load B

Applied loads on a single set of fastenings

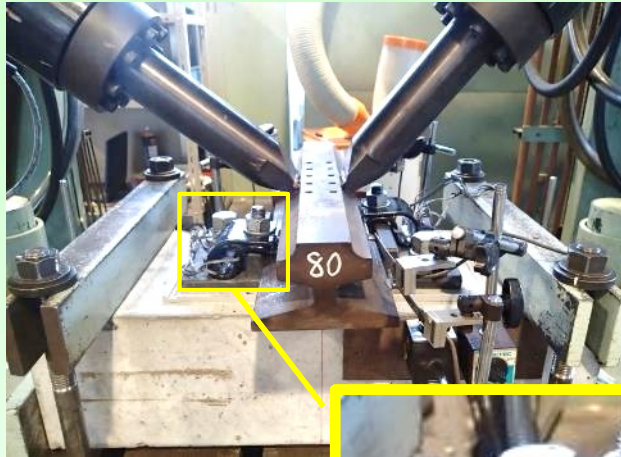
Distributed Load A

Distributed Load B



Inspection of safety against fatigue failure

The following three items are verified by the load test.

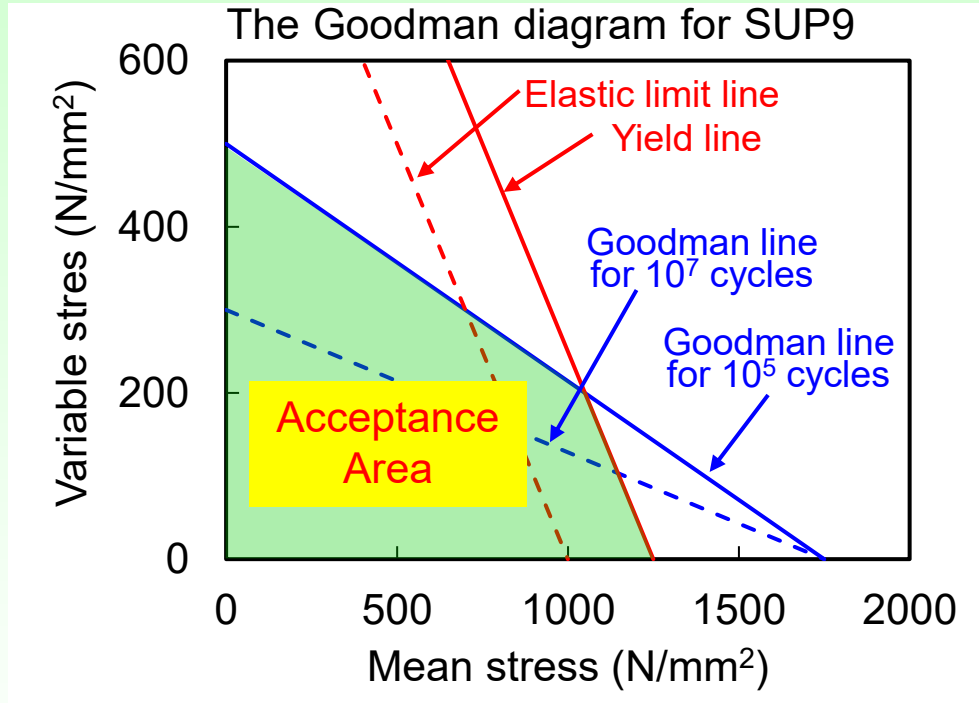
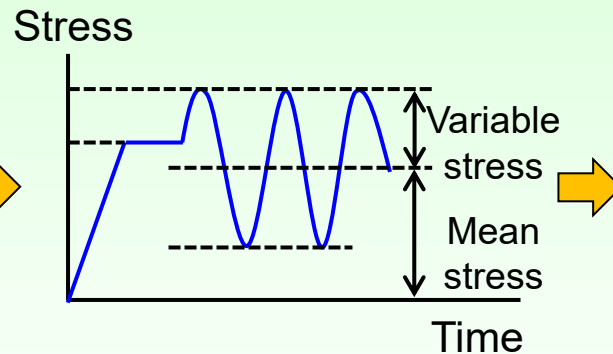


① Rail head lateral **displacement**

② **Stress** of rail clips



Strain gages



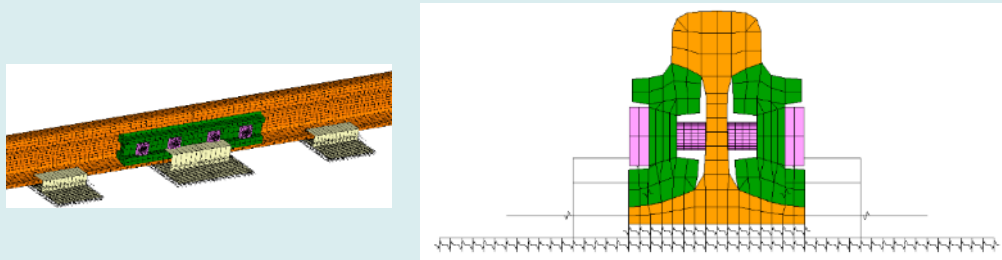
③ The overall **condition** of the rail fastening systems after **1 million load cycles**

Rail fastenings currently used in Japan have passed this inspection.

Recent researches on rail fastening systems

Testing methods for a single set of rail fastening systems **for rail joints**

Establish the FEM model for rail joints



Determination of load conditions for a single set of rail fastenings for rail joints

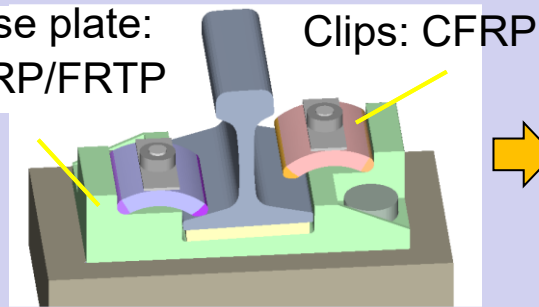


Rail fastenings for rail joints

Examination of rail fastenings with **non-metallic materials** as the main component

Prototype rail fastening systems

Base plate: GFRP/FRTP Clips: CFRP



CFRP: High anti-corrosion
FRTP, GFRP: High electrical insulation

Verified high electrical insulation



Test of electrical insulation

We will continue to take on challenges related to rail fastening systems!

2. Buckling of continuous welded rails

What are continuous welded rails?

In Japan, **continuous welded rails (CWRs)** are defined as rails **without rail joints** and with **a length of 200m or more**.

[Advantages]

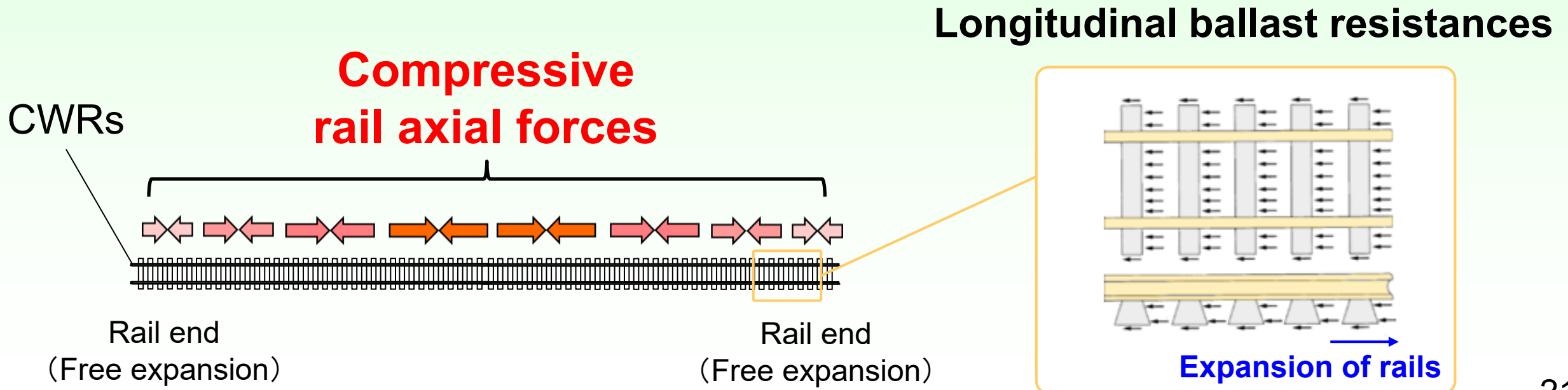
- Improved ride comfort
- Reduced vibrations and noise
- Decreased track maintenance works

Characteristics of CWRs

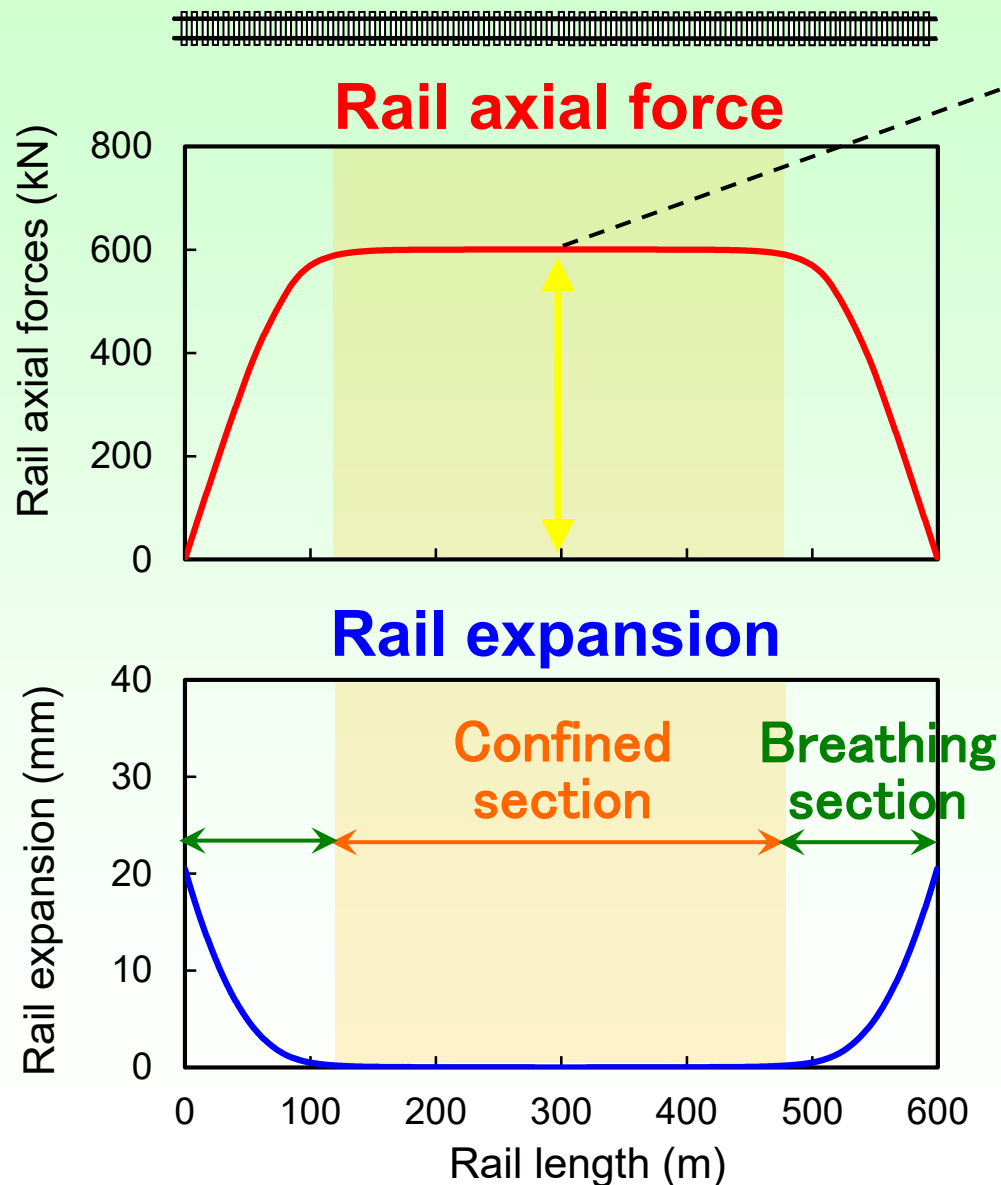
The **expansion of rails** due to temperature increase is **constrained**.



Significant **compressive rail axial forces** are generated.



Rail axial force and rail expansion



$$P = EA\beta\Delta t$$

P : Rail axial force (compressive)

E : Young's modulus

A : Cross sectional area

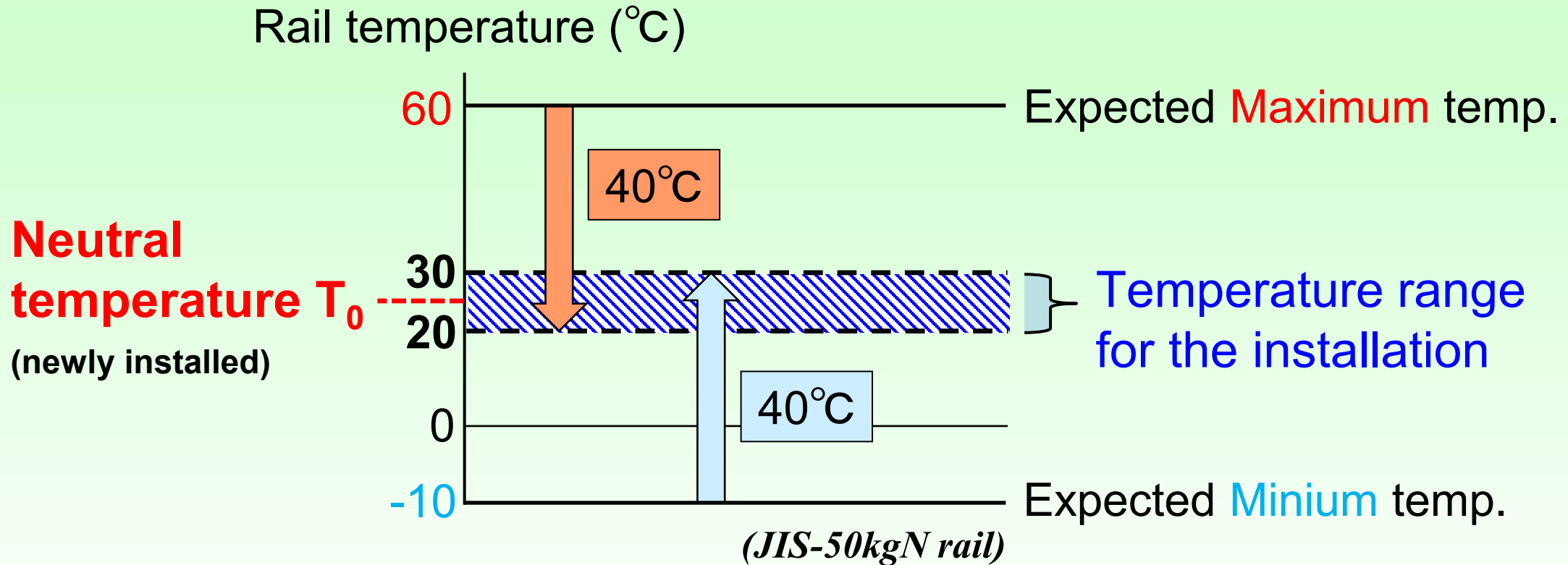
β : Coefficient of thermal expansion

Δt : Temperature increase

from neutral temperature

Rail axial force is zero at both ends of rail, and is maximum in the middle, remaining constant.

Neutral temperature of CWRs newly installed



- New CWRs should be installed **within a specific temperature range.**
- Rail axial force is **zero** at the neutral temperature T_0 .
- Maximum rail axial force depends on **temperature increase from T_0**

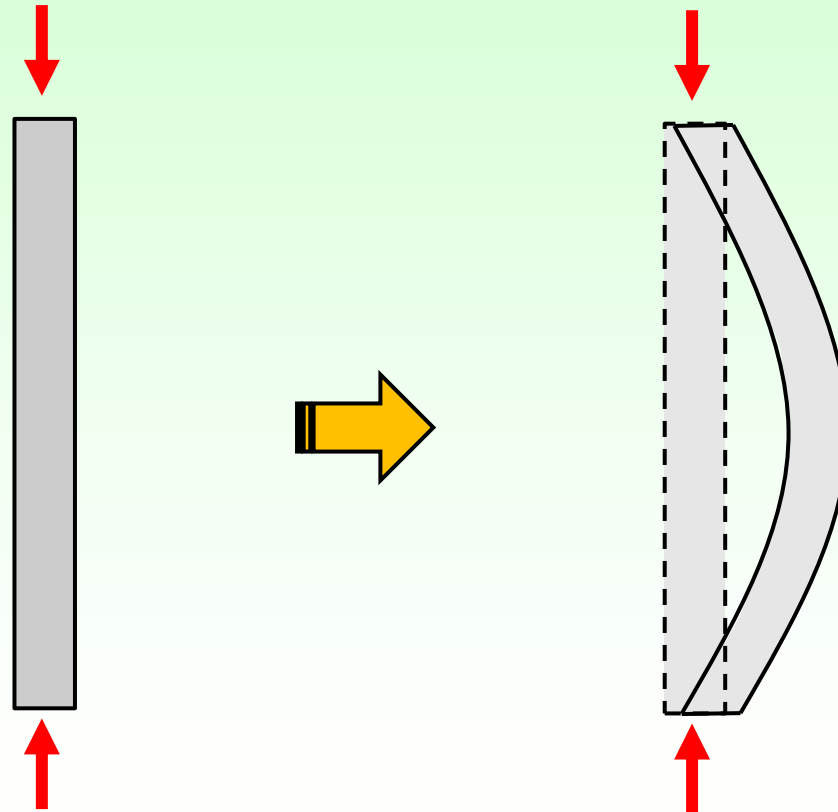
If rail axial force exceeds the limit,
there is a risk of **buckling**



Photo of Buckling Test

What is buckling?

Buckling is the **sudden change in** shape of a structural component subjected to compressive forces.



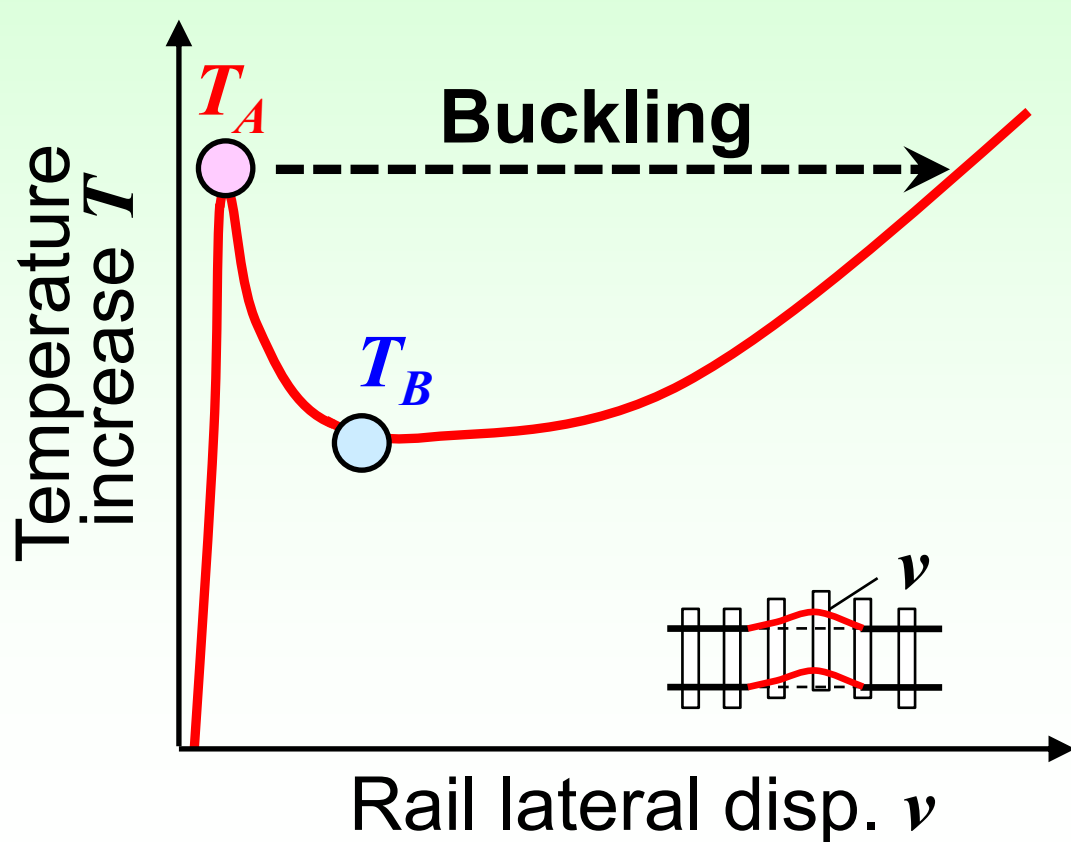
Track buckling test by RTRI (2018)



This study received funding from the MLIT's subsidy aid of railway technology development.

The relationship between rail temperature Increase and rail lateral displacement in track buckling

- ◆ Theoretically, when the rail temperature reaches T_A , buckling occurs.
- ◆ Buckling state does not exist, at values below T_B .



Rail axial force at T_B is defined as
“Minimum buckling strength P_t ”

P_t is used as an **evaluation index**
of buckling strength of CWRs.

Calculations of minimum buckling strength P_t

Currently, the **minimum buckling strength P_t** is calculated by the following equations based on the energy method, in Japan.

$$R \geq R_0$$

$$P_t = 0.987 J^{0.388} g^{0.521}$$

$$R < R_0$$

$$P_t = 1.266 J^{0.374} g^{0.534} - a J^b g_0^c / R$$

$$R_0 = \frac{a J^b g^c}{1.266 J^{0.374} g^{0.534} - 0.987 J^{0.388} g^{0.521}}$$

(JIS-50kgN rail)

P_t : Minimum buckling strength (kN)

J : Parameter by multiplying track panel stiffness with rail lateral stiffness

g : Lateral ballast resistance (kN/m)

r : Longitudinal ballast resistance = $2g$ (kN/m)

R : Radius of curvature (m)

a : Proportional constant, b, c : Exponentiation of constant

Verification of buckling stability of CWRs

Buckling stability is verified by comparing the minimum buckling strength with the maximum rail axial force.

Value of safety level :

$$S = \frac{P_t}{P_{\max}}$$

: Minimum buckling strength (kN)

: Maximum rail axial force (kN)

$$S \geq 1.2$$

Normal state

$$S < 1.2$$

Needs actions

- ◆ Track inspection
- ◆ Restressing of CWRs
- ◆ Reinforcement of lateral ballast resistances, etc.

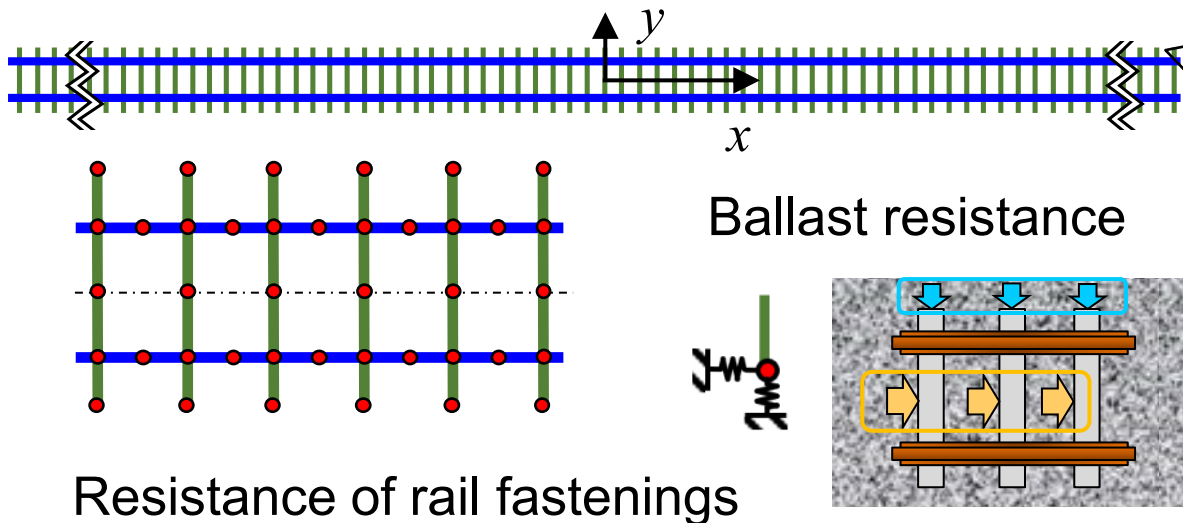
If the safety level falls below the threshold level (1.2), some form of actions are necessary.

Recent researches on buckling of CWRs

Practical method for **estimating buckling temperature of CWRs** based on **lateral track irregularities measured by inspection car**

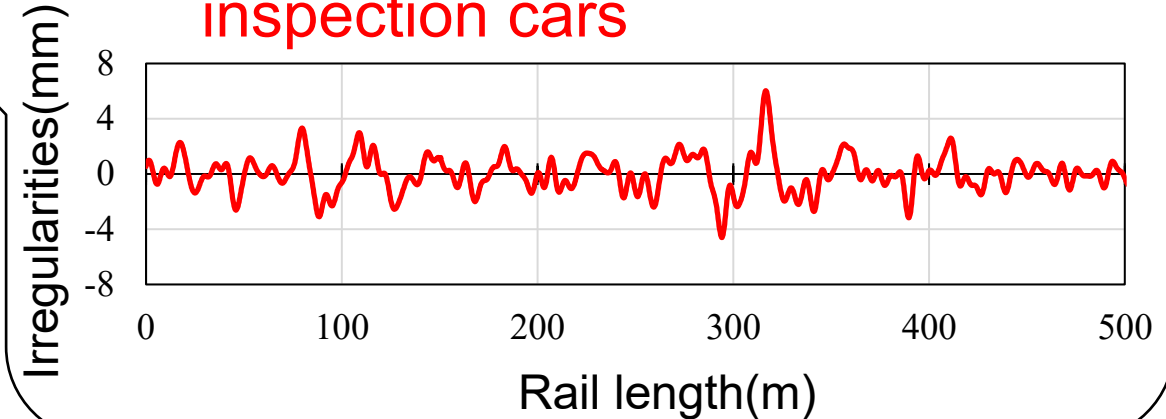
Buckling analysis by FEM using track irregularities measured by inspection cars

FEM model



Resistance of rail fastenings

Lateral track irregularities by inspection cars



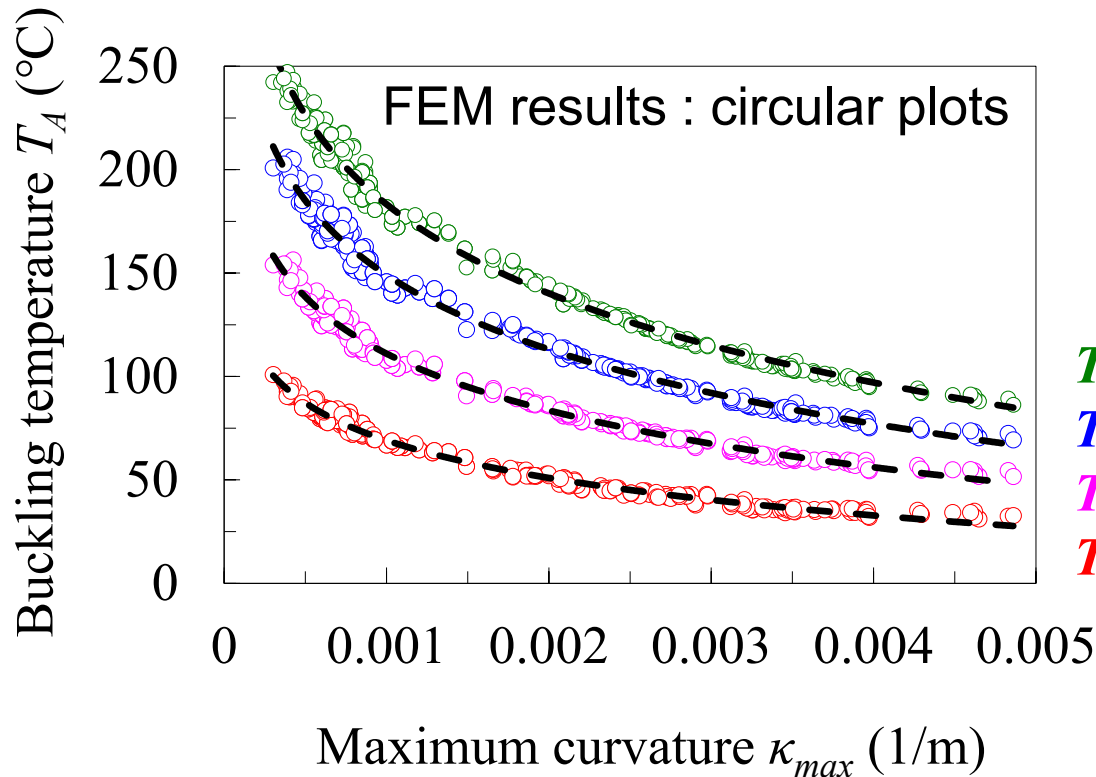
Analysis for over 250 cases



Identifying the key factor that affects track buckling

Recent researches on buckling of CWRs

Proposing formulas to estimate the buckling temperature T_A from the maximum curvature κ_{max} of the lateral track irregularities.



Regression formulas

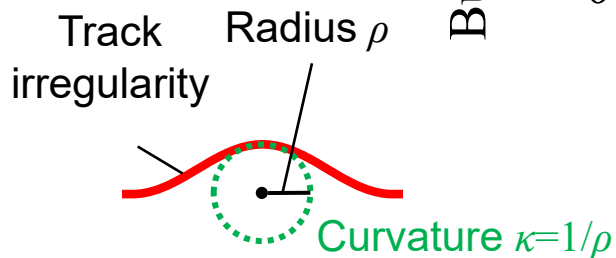
$$T_A = -62.41 \ln(\kappa_{max}) - 247.9 \quad \dots g=8 \text{ kN/m}$$

$$T_A = -52.14 \ln(\kappa_{max}) - 211.2 \quad \dots g=6 \text{ kN/m}$$

$$T_A = -39.75 \ln(\kappa_{max}) - 163.7 \quad \dots g=4 \text{ kN/m}$$

$$T_A = -26.21 \ln(\kappa_{max}) - 112.1 \quad \dots g=2 \text{ kN/m}$$

g : Lateral ballast resistance



We will continue to take on challenges related to CWRs!

Thank you for your attention